

# Hull Girder Ultimate Strength of a Ship Using Nonlinear FE Method

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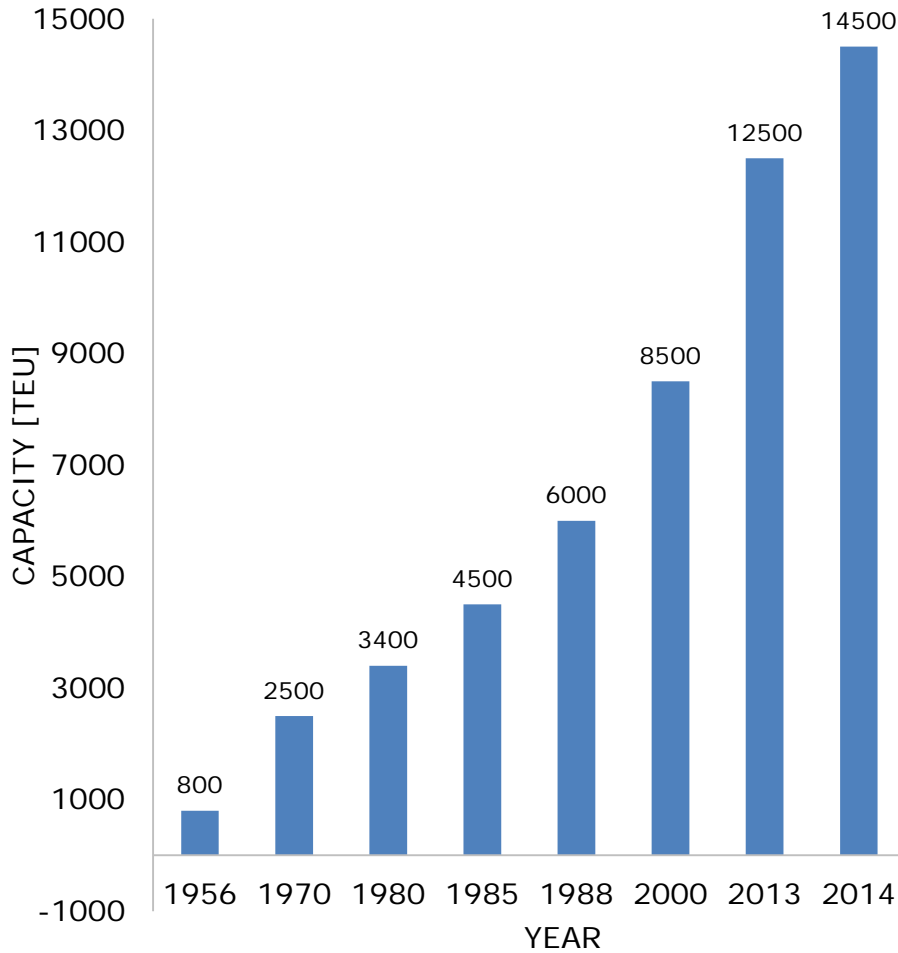
Presented by

Hemanand Kalyanasundaram

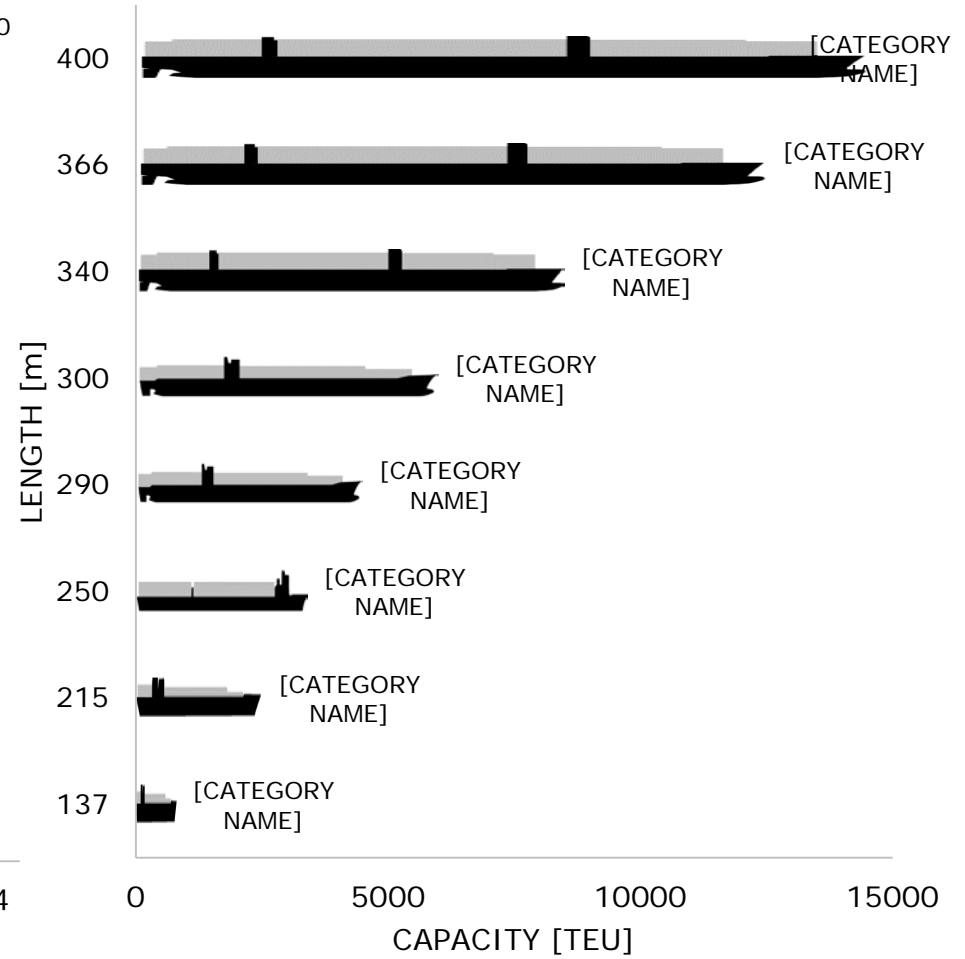


# Evolution of Container ships

## Capacity of container vessels

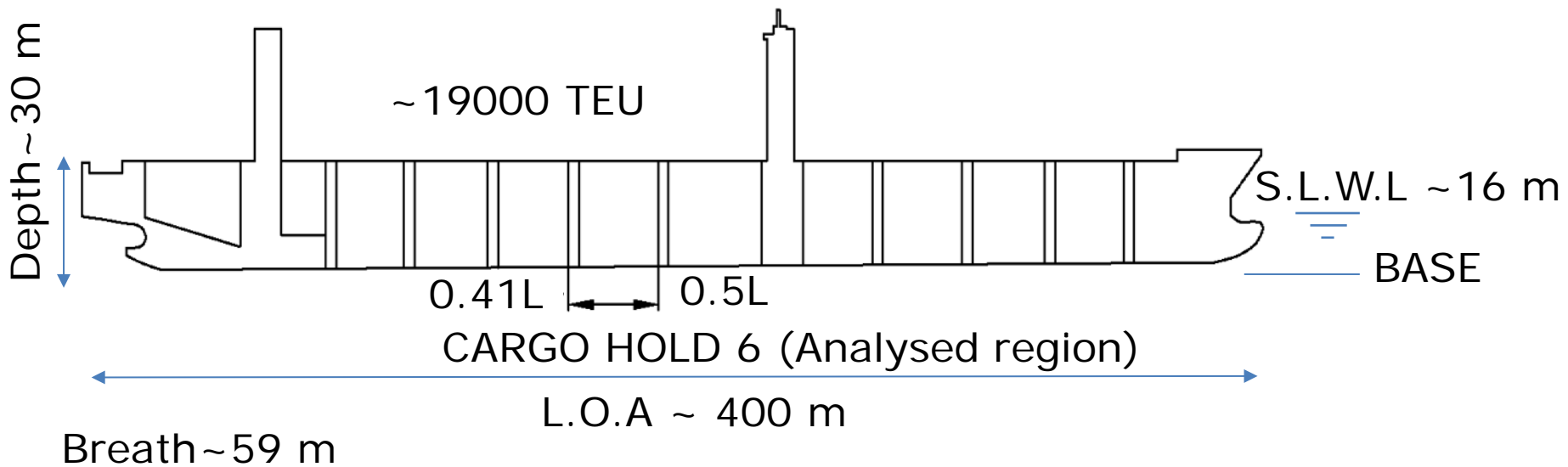
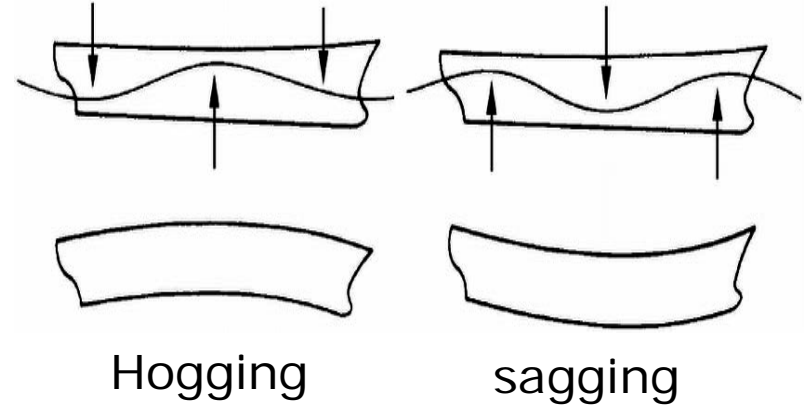


## Length of ship



Reference: <https://people.hofstra.edu/geotrans/eng/ch3en/conc3en/containerhips.html>

# Motivation and parameters of analyzed ship



Reference:

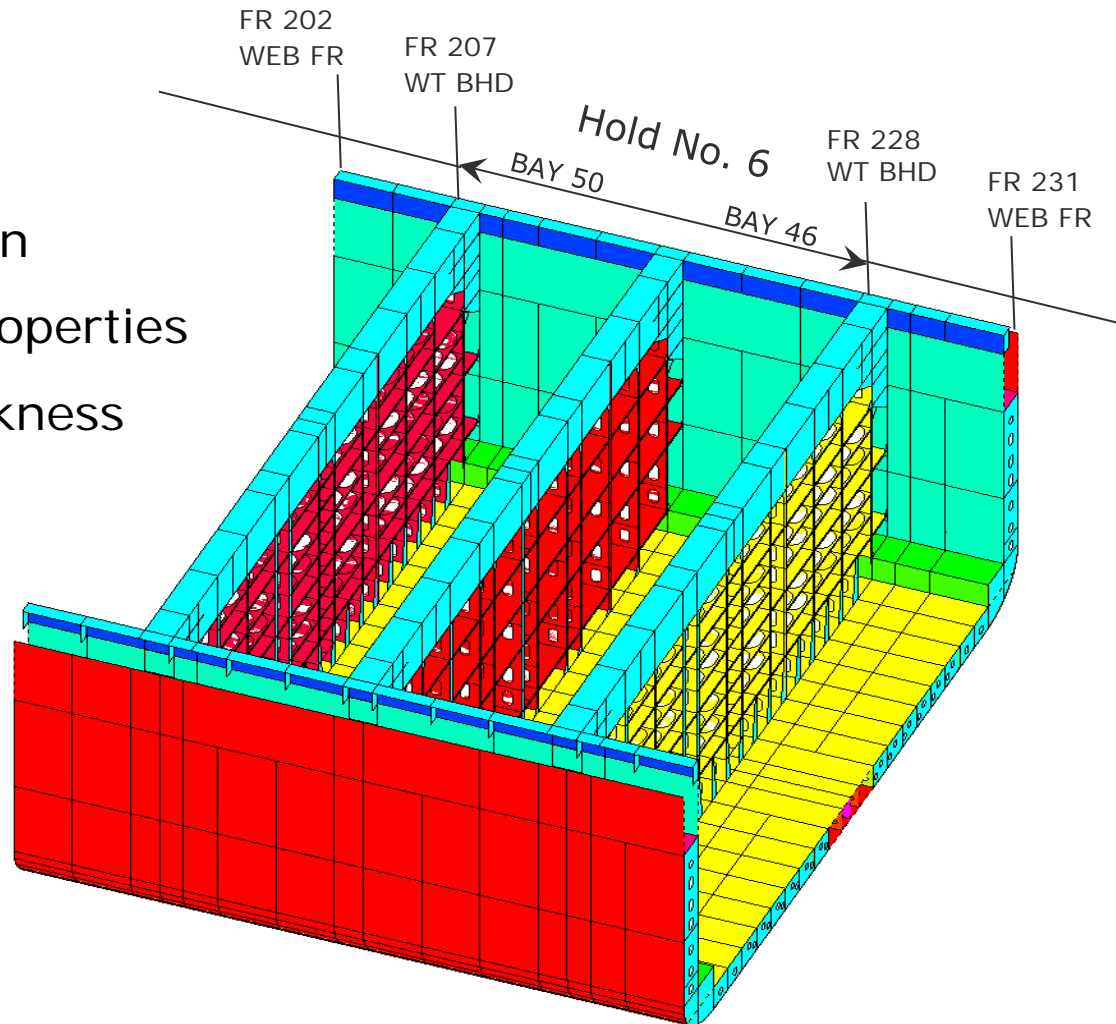
<http://gcaptain.com/mol-comfort-incident-photos/>

[https://www.researchgate.net/figure/228891097\\_fig2\\_Figure-2-Hogging-and-sagging-of-ship-hull](https://www.researchgate.net/figure/228891097_fig2_Figure-2-Hogging-and-sagging-of-ship-hull)

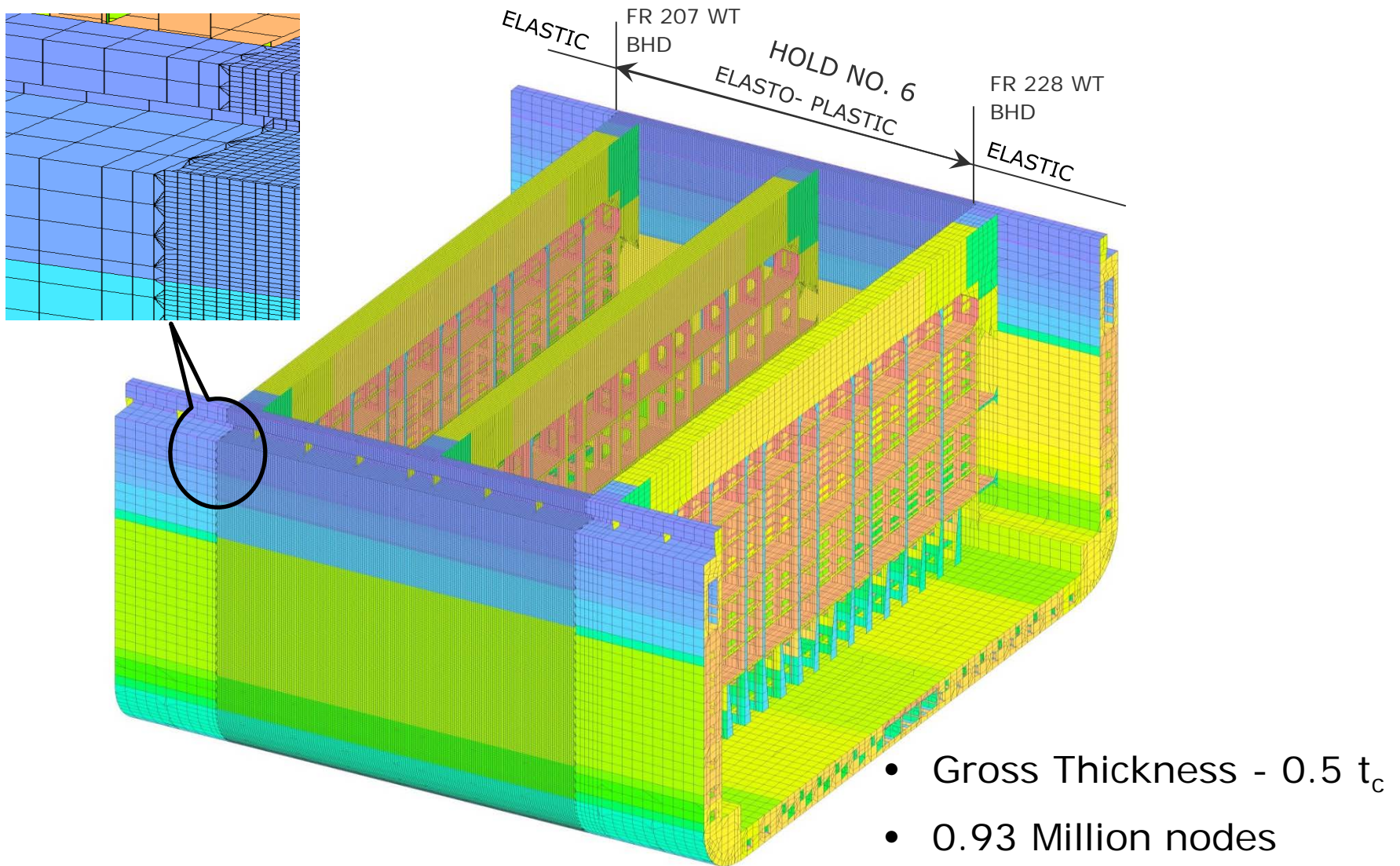
1-FEB-2017

# CAD model preparation

- GL Poseidon
- Material properties
- Gross Thickness
- Cutouts

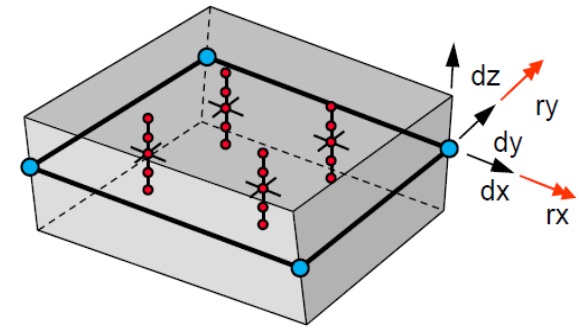
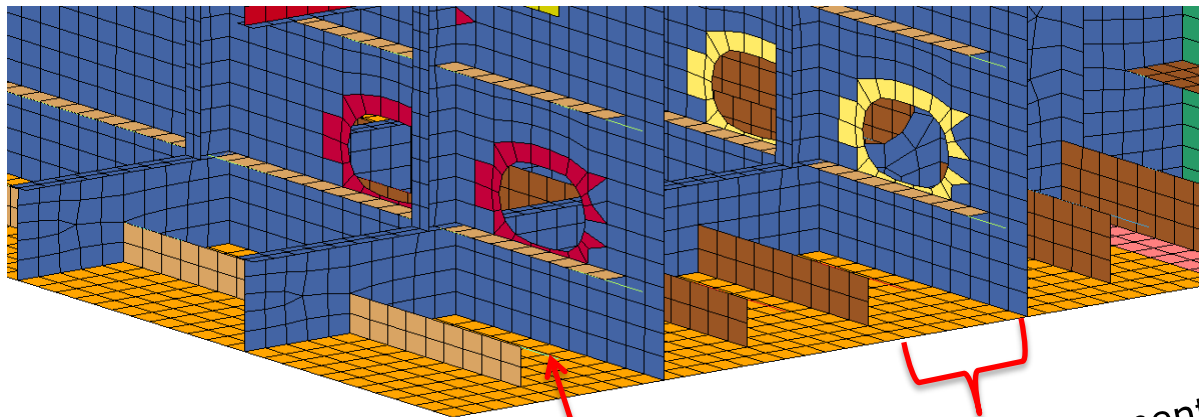


# FE model preparation – Cargo hold model





# Mesh size and considering nonlinearities



Shell 16

BEAM Element

5 to 6 Elements

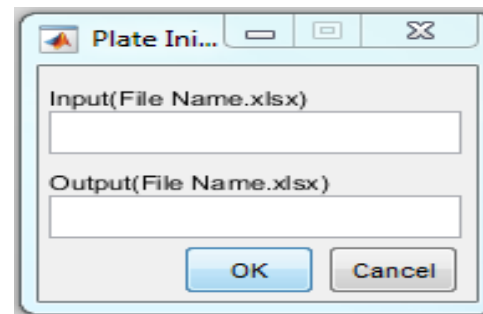
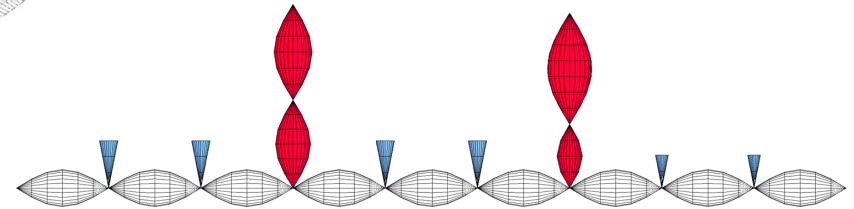
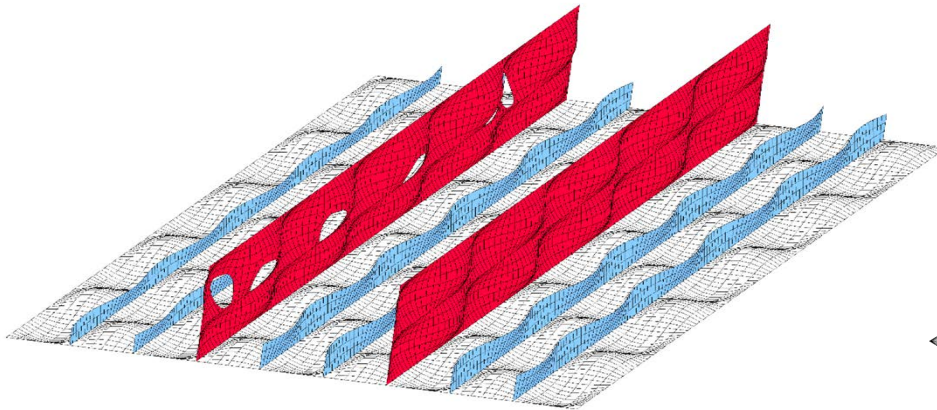
Nonlinearities	Analysed area	Other
Geometric	Shell 16 (plates + stiffener web)	Shell 2 (plates)
	Beam (Stiffener flange)	Truss (stiffener)
Material	Elasto-plastic (Bilinear model) $E_t = 1000 \text{ Mpa}$	Elastic

Reference:

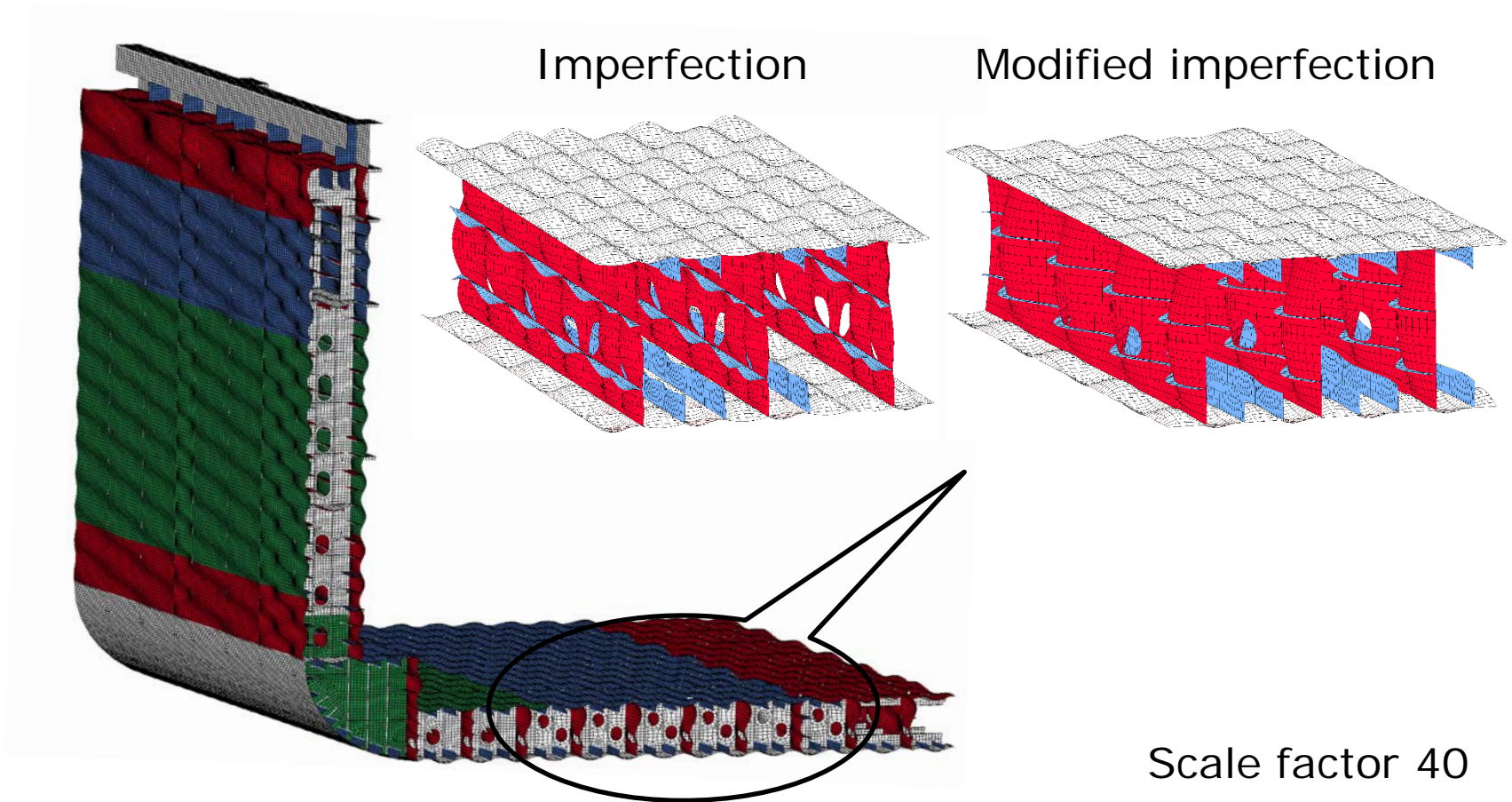
<https://www.dynamore.de/de/download/papers/2013-Is-dyna-forum/documents/review-of-shell-element-formulations>

# Modeling geometrical initial imperfections

- Maximum deflection  $\delta_{plate} = \frac{b}{200}$      $\delta_{stiffweb} = \frac{a}{1000}$ 
  - a Frame spacing
  - b Stiffener spacing

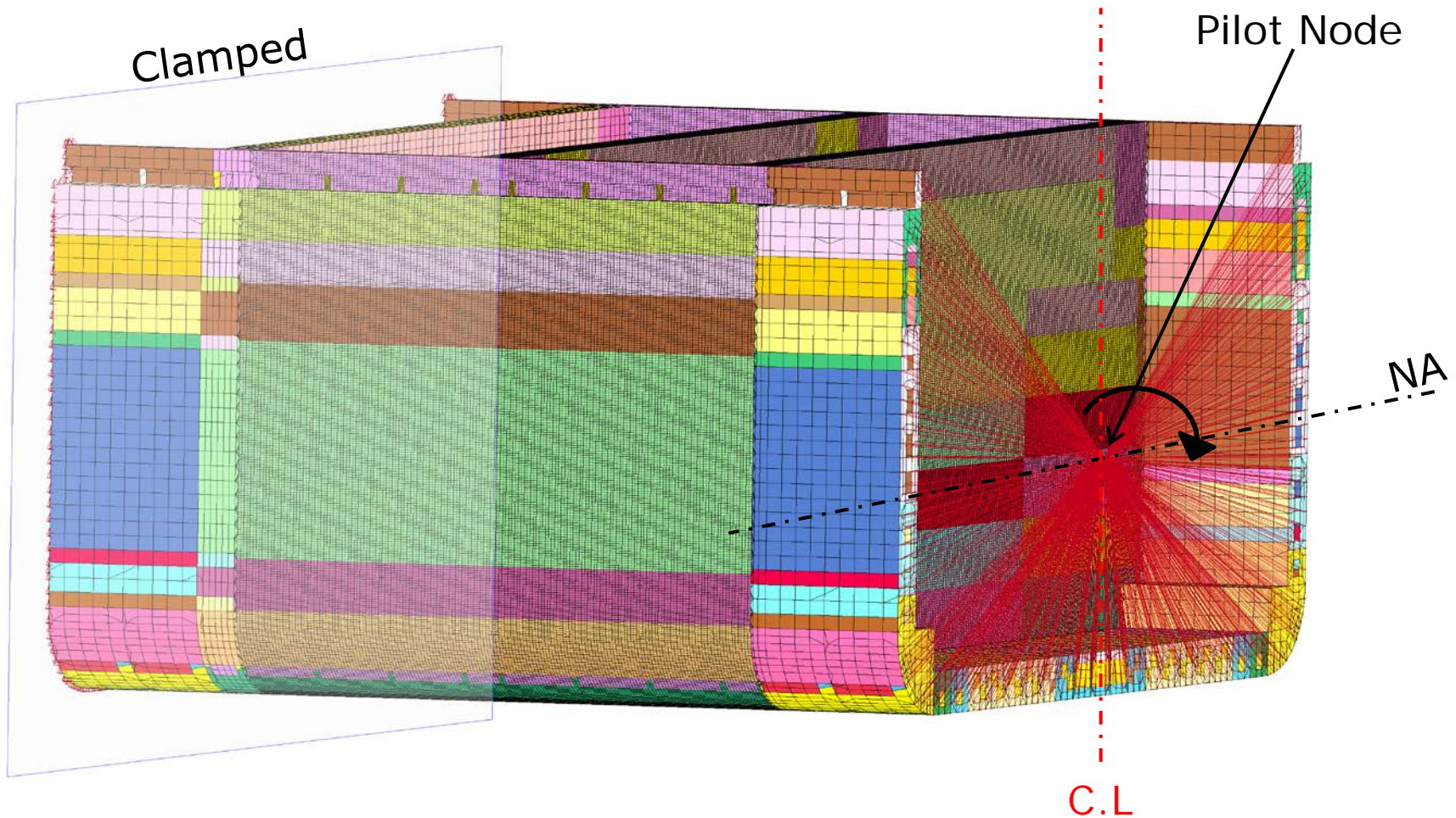


# Imperfection on FE model - Section view





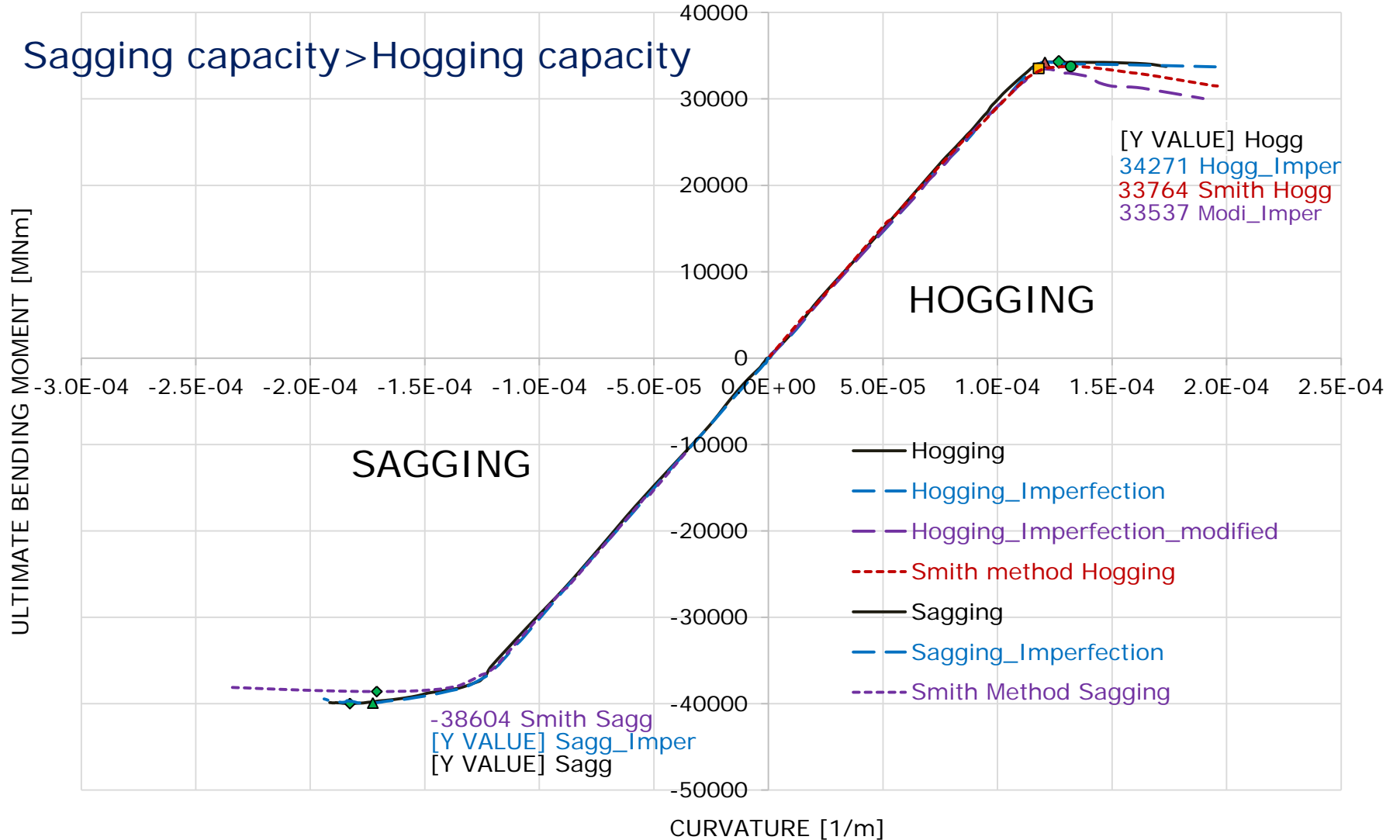
# Cargo hold model – Boundary condition and analysis



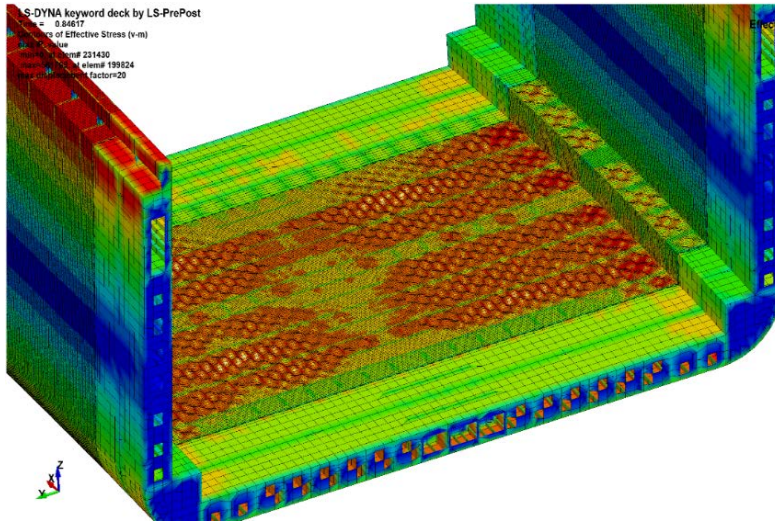
Nonlinear analysis is solved implicitly using BFGS method in LS-DYNA

# Results of cargo hold analysis

## Ult. bending moment capacity -Pure Vertical Moment



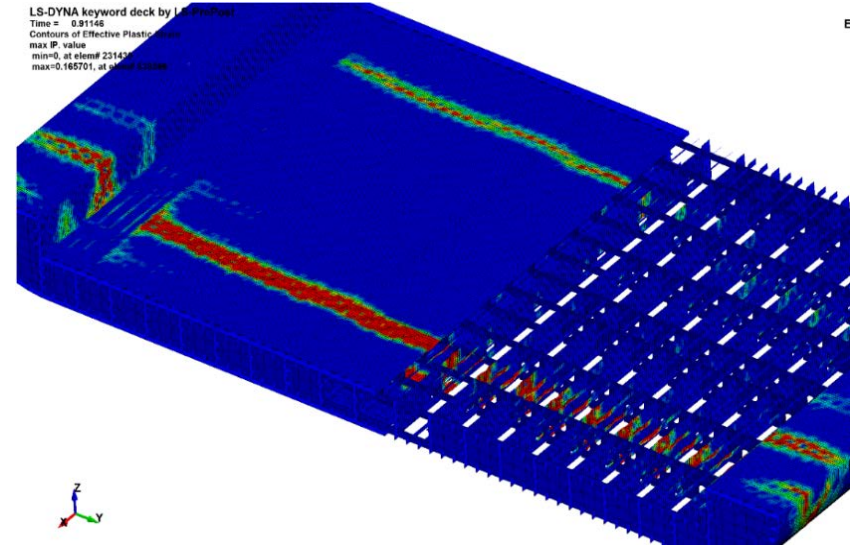
# Results under hogging (without imperfection model)



vonMises @M=33228MNm

Scale factor= 20

Deflection follows modeled imperfection



Plastic strain propagation Mu

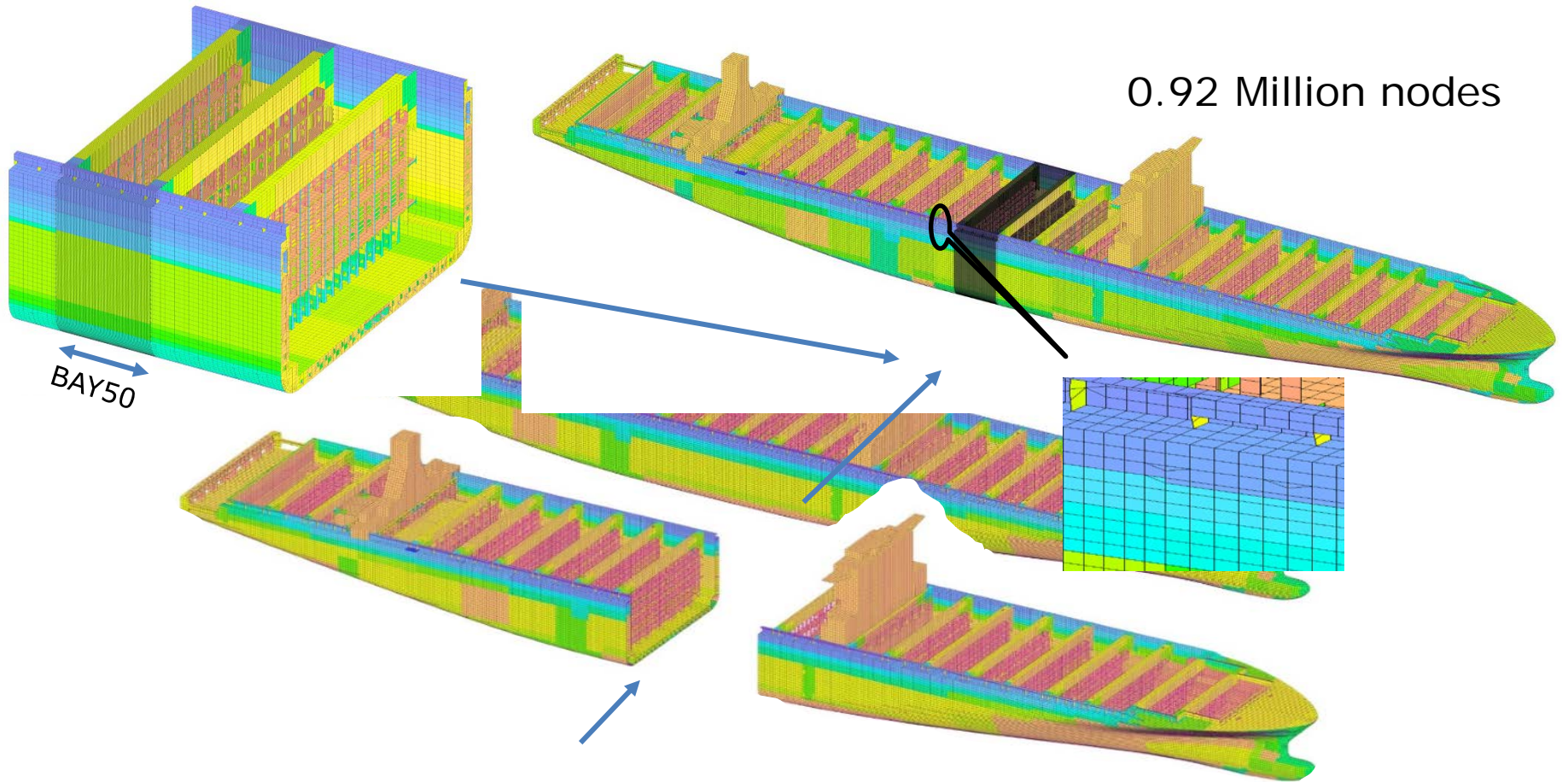
Scale Factor=1

$$\frac{M_{u-hogg}}{\gamma_M * \gamma_{DB}} = \frac{33764}{1.1 * 1.1} = 27904 \text{ MNm}$$

- Hogging condition is critical.
- Smith method with safety factor 20% conservative.



# Preparation of global FE model



0.92 Million nodes

BAY50

Mesh size of ~800 mm

# Loading generation for Global FE analysis

- GL Shipload software for load generation.
- Static loading condition – Still water loadings
  - 90% of Max. permissible stillwater BM - 1.04xE7 KNm [UR S11 A]
  - Draft - 14.5 m

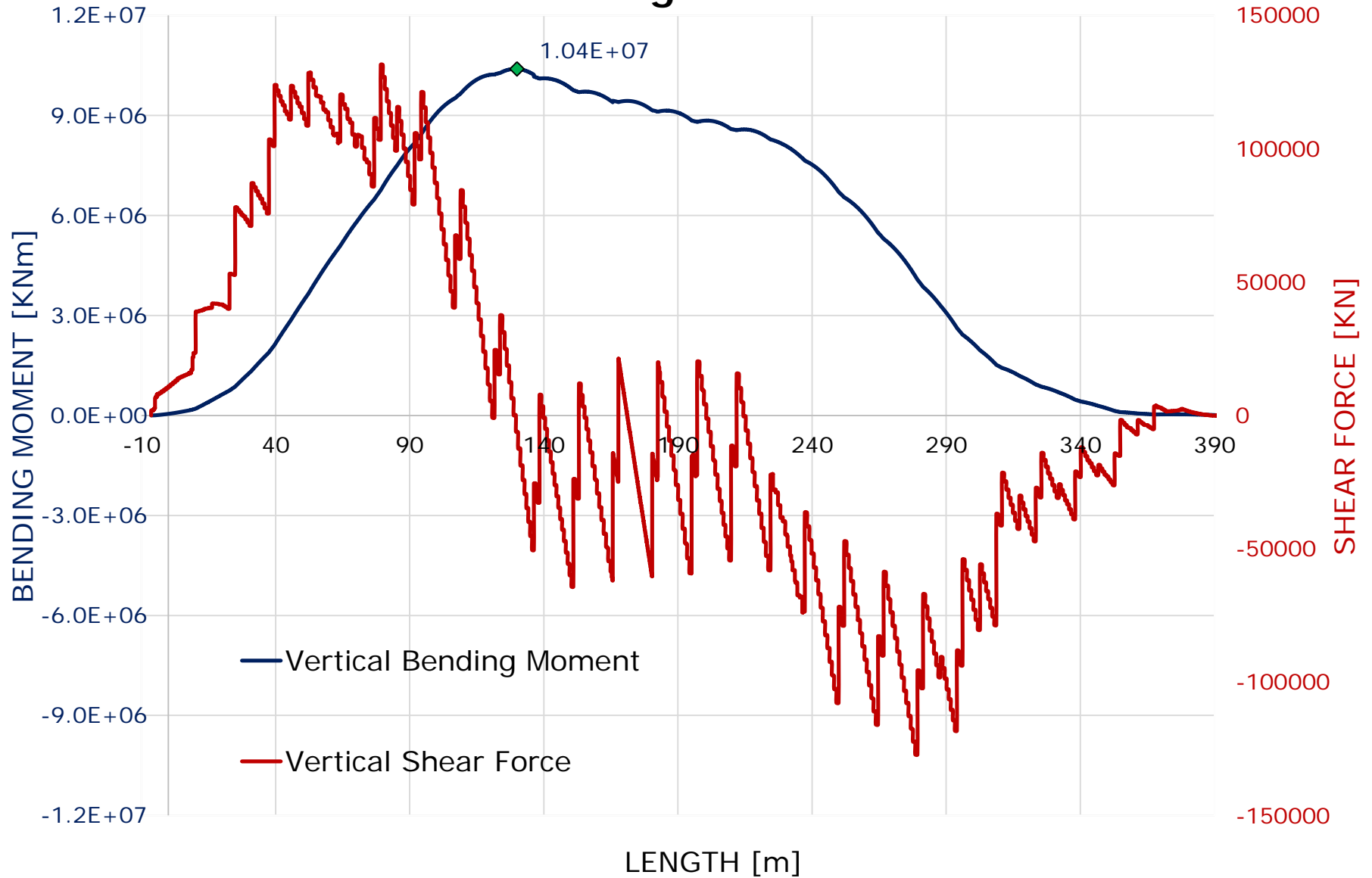
Analysed area Cargo hold 6	
Container loadings	4580 Tonnes/Bay
Bottom balast tanks	60% Fill rate
Side balast tanks	20% Fill rate

- Dynamic loadings - Waves loadings

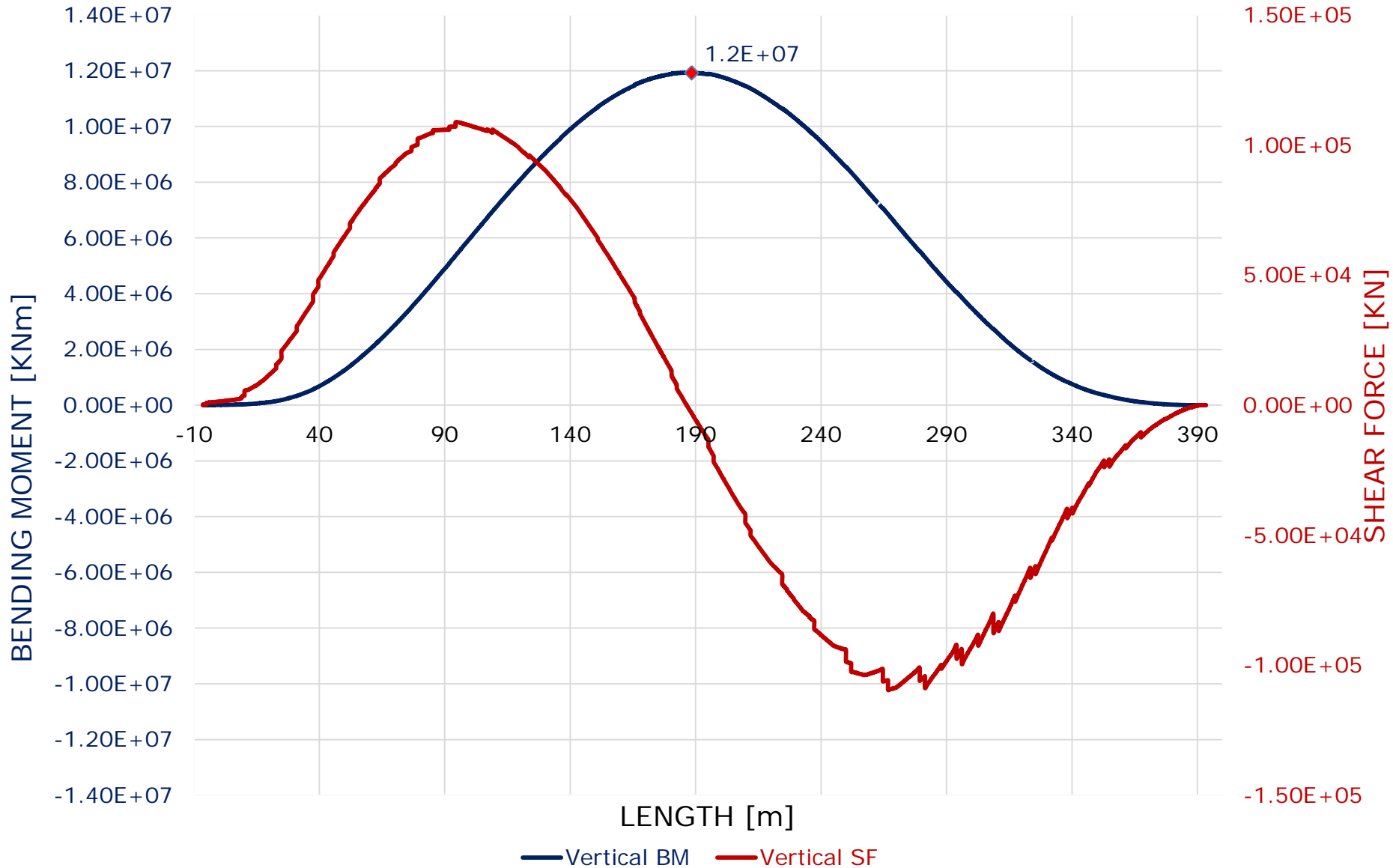
Dynamic Loading Cases	Wave Amplitude [m]	Wave Direction	Ship Velocity [Knots]
Head Sea	6.5	180°	5
Oblique Sea	4.7	120°	5



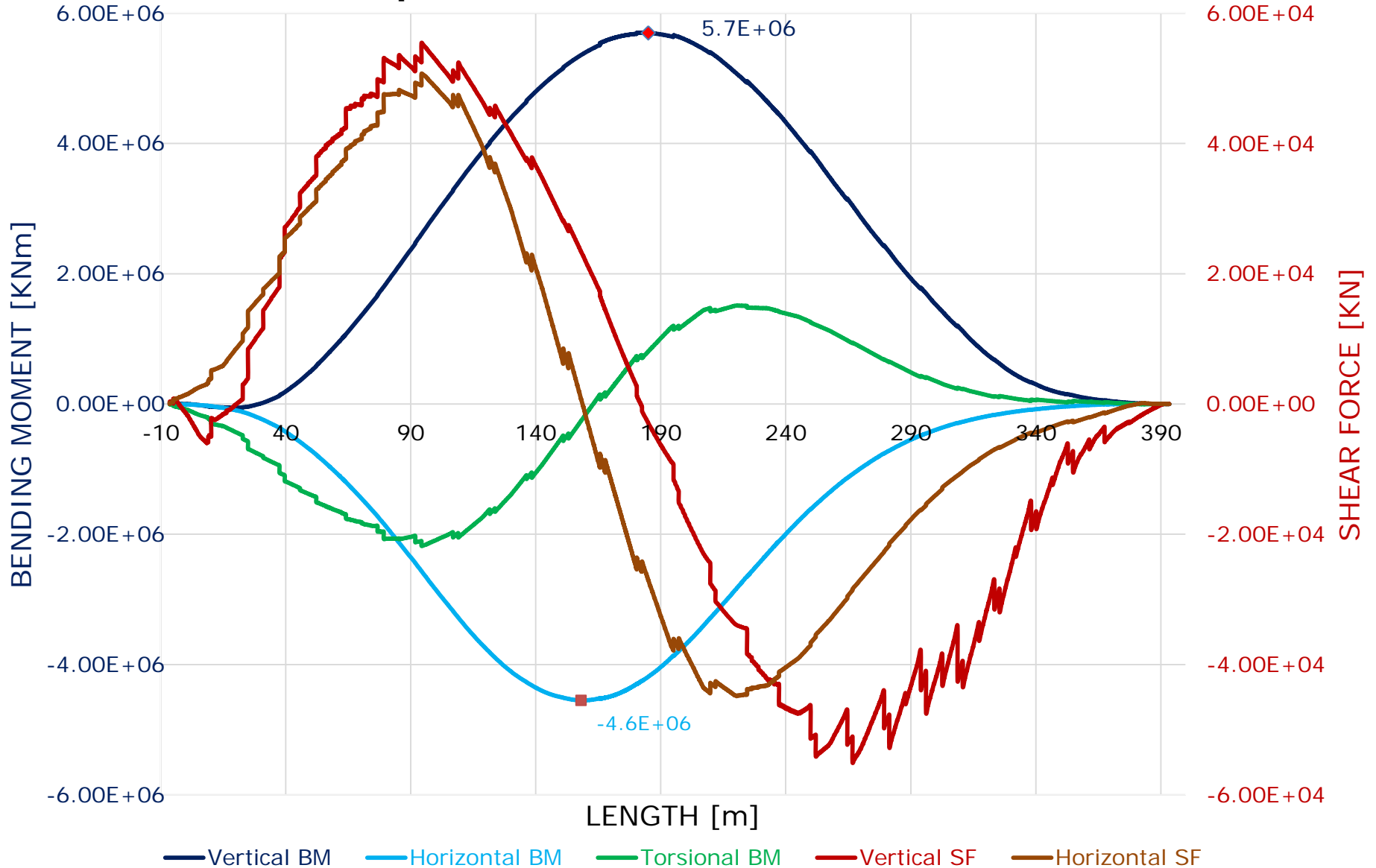
# Static loading condition



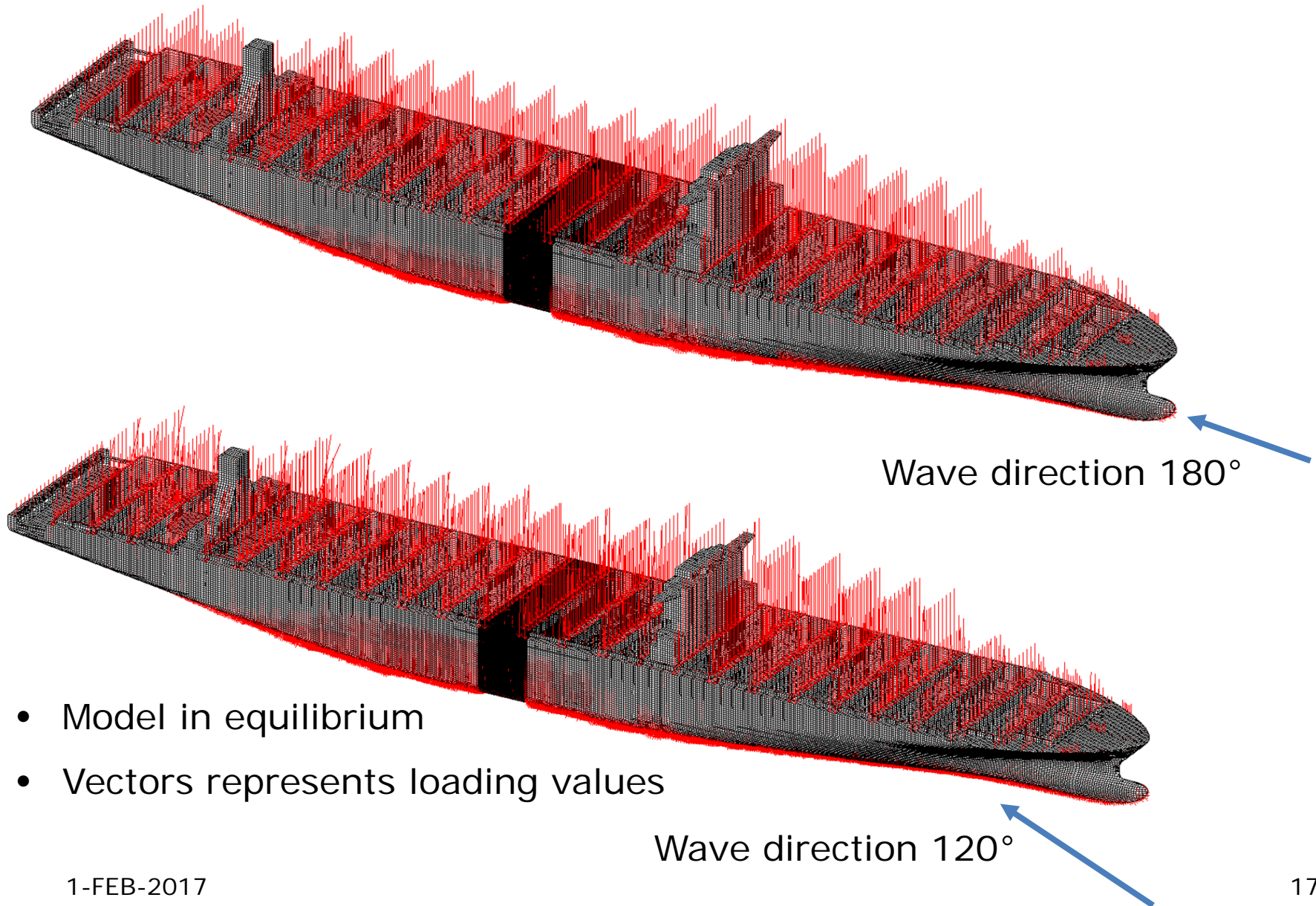
# Head sea case-Maximum hogging



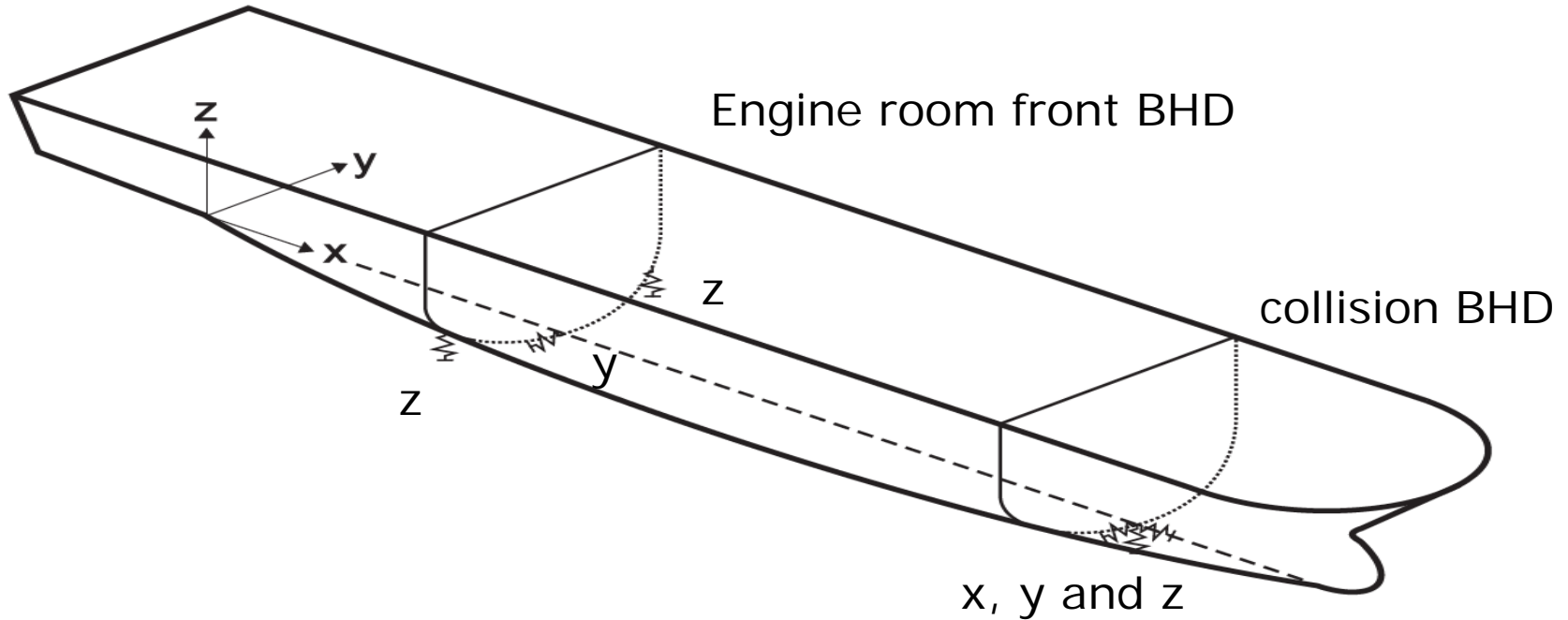
# Oblique sea case- Combined loads



# Nodal loads on global FE model



# Boundary conditions - Global analysis

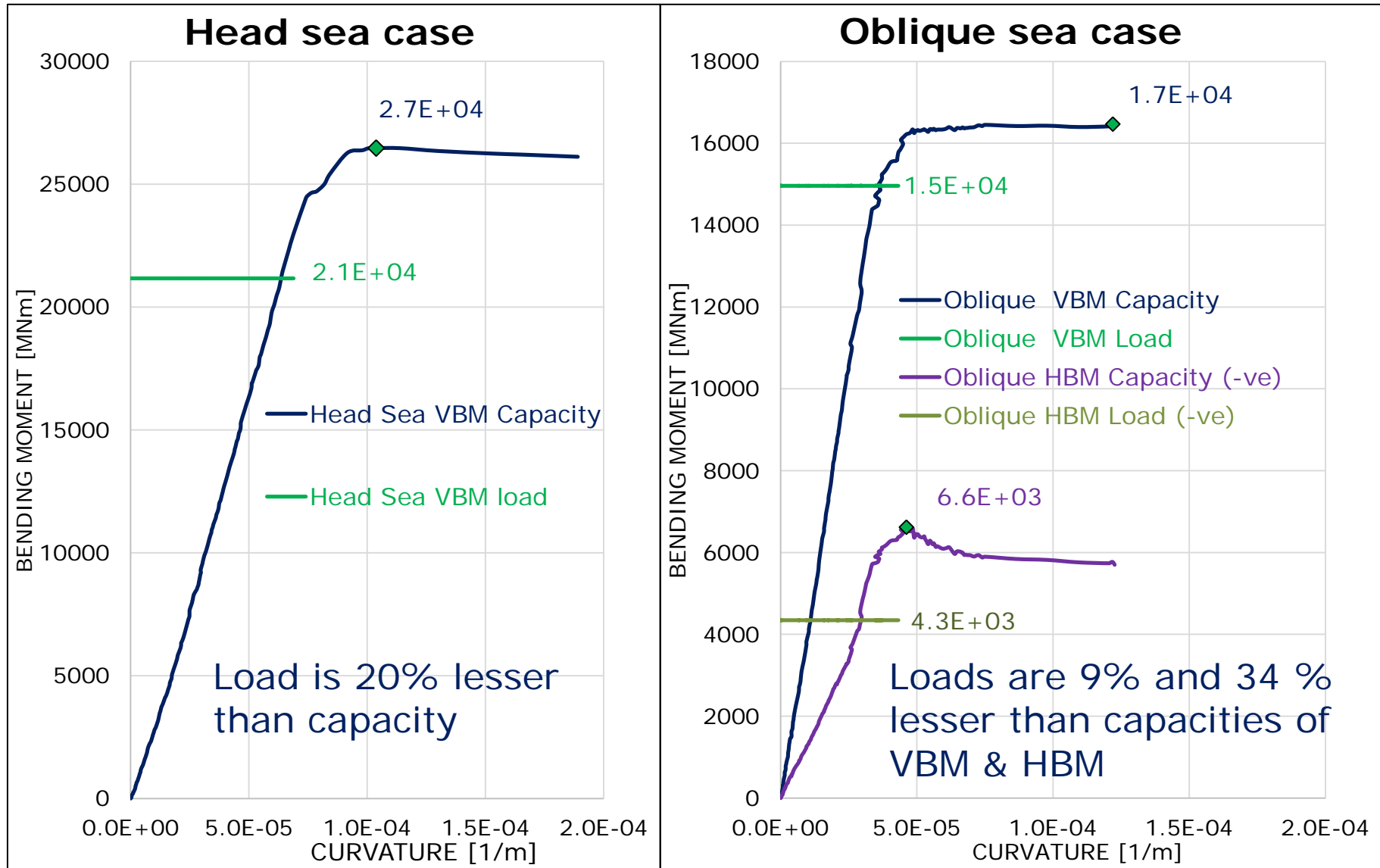


Nonlinear analysis is solved implicitly using BFGS method in LS-DYNA

Reference:  
DNV GL Class guidelines, Finite Element Analysis.



# Results of global FE analysis-Ult. bending moment



## Conclusion

- Hogging condition is critical in container vessels.
- Smith method provides conservative results for pure VBM.
- Designed container ship is safe under combined loading conditions.